MECHANICAL BEHAVIOR OF I-BEAMS REINFORCED BY UNIDIRECTIONAL CARBON FIBRE, UNIDIRECTIONAL GLASS FIBRE AND CARBON FIBRE LAMINATES

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Abstract. Industry demands new engineering and material solutions. One of these solutions are fibre reinforced polymers. They are light and strong for application as a material for I-beams. An I-beam is the best section for a homogeneous material because of the highest resistance moment. The popularity of composite materials introduces wide use in most branches of engineering and mostly as fibre reinforced polymers (FRP). This paper presents numerical and analytical studies on the mechanical behaviour of the I-beams made of fibre reinforced polymers reinforced by glass fibres (GFRP) and carbon fibres (CFRP) comparing to the structural steel S235JR. Five I-beams with different composite structures and one steel I-beam were numerically tested. Four-point test according to ASTM C 78 – 02 was conducted. Numerical simulation made in SOLIDWORKS software in the Static simulation mode was used. The obtained results were analysed and an attempt to determine the optimal parameters for combination of different composite materials was conducted. As a result of numerical analysis values of deflection and normal stress were obtained for polymer I-beam reinforced by glass and carbon fibres comparing to the steel I-beam.

Keywords: carbon fibre, fibre reinforced polymer, glass fibre, I-beam, finite element method.

Introduction

Fibre reinforced polymer (FRP) composite beams are becoming very popular and used often in different engineering areas. Comparing to the mostly used traditional materials such as steel or wood, fibre reinforced polymer beams are more corrosion resistant, have advantageous weight-to-stiffness and weight-to-strength ratio and could be adjusted for exact application what makes them very flexible in terms of application [1-5].

Fibre reinforced polymer composite beams are already successfully used in fields like automotive, civil, naval etc. engineering [6]. Mainly composite beams are designed depending on a specific task, what is exactly their purpose. For example, the FRP pedestrian bridge built in Okinawa or Tainan is made of specific continuous girders [7-9]. It is difficult to unify cross-section parameters, composite material parameters and thus makes it difficult to replace already existing steel structural I-beams. GFRP and CFRP beams are lighter than steel beams. Though GFRP is much cheaper than CFRP, it has smaller stiffness than CFRP and steel [7; 10]. Therefore, there are difficulties in usage of GFRP materials for heavy duty bridges and constructions due to the deflections. One of the ways to overcome this problem, is to use additional reinforcement in glass fibre reinforced polymer, to use carbon fibre reinforced polymer instead of glass carbon reinforced polymer. It can help architects make better constructions and give more freedom in shaping.

Novelty of this work is an attempt to find an optimal combination material and its parameters using the numerical study finite element method for the shape of a standard I-beam cross-section which is the best section for a homogeneous material because of the highest resistance moment.

Materials and methods of the simulation

The experiment results were obtained by the finite element method using computer simulation on SOLIDWORKS software. The I-beam with cross-section HE100B by Euronorm 53-62 standard was used [11;12].

The cross-section size characteristics are shown below in Table 1. For the fully laminated composite I-beam was used a model made from laminates with difference in fillet radius. For the laminated I-beam the fillet radius is zero. The I-beam was tested by static simulation mode. Comparison of the results of the four-point test was made. Four-point test was conducted according to the ASTM C 78 – 02 standard [13]. The setup scheme for the four-point test is shown in Figure 1.
Table 1

<table>
<thead>
<tr>
<th>Number of beam</th>
<th>Type of beam</th>
<th>Cross-section</th>
<th>Material</th>
</tr>
</thead>
<tbody>
<tr>
<td>1</td>
<td>S235JR</td>
<td></td>
<td>S235JR Structural steel</td>
</tr>
<tr>
<td>2</td>
<td>GF40%</td>
<td></td>
<td>40% reinforced GRFP Unidirectional</td>
</tr>
<tr>
<td>3</td>
<td>CF40%</td>
<td></td>
<td>40% reinforced CRFP Unidirectional</td>
</tr>
<tr>
<td>4</td>
<td>CFLW40% Unidirectional laminas</td>
<td></td>
<td>40% reinforced CRFP Lamiated by woven [0^\circ/90^\circ/0^\circ/90^\circ/0^\circ] flanges and [0^\circ/90^\circ/0^\circ] web. Each ply is 2 mm</td>
</tr>
<tr>
<td>5</td>
<td>CFLW40% Bidirectional laminas</td>
<td></td>
<td>40% reinforced CRFP Lamiated by woven [0^\circ/90^\circ/-45^\circ/45^\circ/0^\circ/90^\circ/45^\circ/-45^\circ/90^\circ/0^\circ] flanges and [0^\circ/45^\circ/90^\circ/90^\circ/45^\circ/0^\circ] web. Each ply is 2 mm and woven</td>
</tr>
<tr>
<td>6</td>
<td>CF40% + CFLW Bidirectional laminas</td>
<td></td>
<td>40% reinforced CRFP Unidirectional Lamiated by woven reinforcement [0^\circ/90^\circ/90^\circ/0^\circ] flanges and unidirectional web. Each ply is 2 mm and woven</td>
</tr>
</tbody>
</table>

Fig. 1. Setup scheme for the four-point test (ASTM C 78 – 02)

As it was described above, six different materials and different configurations of them were used for the testing. The test was done using the finite element method on Solidworks simulation software static simulation mode [14–16]. The left cross-section of the beam translationally fixed and free in rotation. The right part translationally fixed only in X axis and Y axis directions and free in rotation,
while in Z axis direction it is free to move. Two forces applied each 10 000 N. Total length of the testing beam is 2000 mm.

![SOLIDWORKS setup](image)

Fig. 2. SOLIDWORKS setup: 1 - fixed edge in X, Y, Z directions; 2 - fixed edge in X, Y directions; 3 - applied forces each 10 000 N.

The setup scheme is shown in Figures 1 and 2. At the cross-section 1 in Figure 2 the lower edge of the lower flange is translationally fixed in X, Y, and Z directions but free in rotation. At the cross-section 2 in Figure 2 the lower edge of the lower flange is translationally fixed only in X and Y directions and free in rotation. At the cross-section 3 in Figure 2 two forces are applied each 10 000 N on the upper edge of the upper flange.

Six different materials and different configurations of them for the same geometry cross-section I-beams were tested. The properties of materials are given in references [17-22]. Knowing the mechanical properties of each constituent material of the composite material, it is possible to find the properties of the 40% reinforced composite material properties using the rule of mixture (1) and Halpin-Tsai (2) equations [23; 24]. In this case, the woven composite material will be modelled as a 2-layer unidirectional composite material. The rule of mixtures (1) will be used to find the Young’s modulus along the reinforcement direction.

\[
\frac{E_{c1}}{1 - V_m} = E_f \cdot V_f + E_m \cdot V_m,
\]

where \( E_{c1} \) - Young’s modulus longitudinally of composite, Pa; \( E_f \) - Young’s modulus longitudinally of fibre, Pa; \( V_f \) - volume fraction of the fibre, m³; \( V_m \) - volume fraction of the matrix, m³.

To calculate in the transverse direction, Halpin-Tsai equations are used.

\[
\frac{E_{c2}}{1 - V_m} = E_{ml} \cdot \frac{1 + \xi_1 \cdot \eta_1 \cdot V_f}{1 - \eta_1 \cdot V_f},
\]

where \( E_{c2} \) - Young’s modulus transversally of composite, Pa; \( E_{ml} \) - Young’s modulus longitudinally of matrix (\( E_{ml} = E_{m2} \)), Pa; \( \eta_1 \) - Halpin-Tsai coefficient, Unitless. \( \xi_1 \) - Halpin-Tsai coefficient (at \( \xi_1 = 2 \) gives accurate \( E_{c2} \) values), Unitless.

\( \eta_1 \) from equation (2) and \( \eta_2 \) is found by formulas (3) and (4).

\[
\eta_1 = \frac{E_{f1} - E_{m1}}{E_{f1} + \xi_1 \cdot E_{m1}},
\]

\[
\eta_2 = \frac{G_{f12} - G_{m12}}{G_{f12} + \xi_2 \cdot G_{m12}},
\]

where \( \eta_2 \) - Halpin-Tsai coefficient, Unitless; \( \xi_2 \) - Halpin-Tsai coefficient (at \( \xi_2 = 1 \) gives accurate \( E_{c2} \) values), Unitless; \( G_{f12} \) - Shear modulus longitudinally of fibre, Pa; \( G_{m12} \) - Shear modulus of matrix longitudinally (\( G_{m12} = G_{m23} \)), Pa.

Shear modulus and Poisson’s ratios are found by formulas (5) and (6).
\[ G_{c12} = G_{m12} \cdot \frac{1 + \nu_2 \cdot \eta_1 \cdot V_f}{1 - \eta_1 \cdot V_f}, \] (5)

where \( G_{c12} \) – shear modulus longitudinally of composite \((G_{c12} = G_{c23})\), Pa.

Shear modulus and Poisson’s ratios are found by formulas (5) and (6).

\[ \nu_{c21} = \frac{E_{c2}}{E_{c1}} \cdot \nu_{c12}, \] (6)

where \( \nu_{c21} \) – Poisson’s ratio transversally of composite, Unitless;
\( \nu_{c12} \) – Poison’s ratio longitudinally of composite, Unitless.

After finding these values, it is possible to start simulation of the materials. For the glass fibre composite, the same steps were applied. The obtained results are shown in Table 1.

### Table 1

<table>
<thead>
<tr>
<th>Mechanical properties</th>
<th>Units</th>
<th>40% reinforced CRFP</th>
<th>40% reinforced GRFP</th>
</tr>
</thead>
<tbody>
<tr>
<td>Young’s modulus</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>E11</td>
<td>GPa</td>
<td>94.4</td>
<td>37.2</td>
</tr>
<tr>
<td>E22</td>
<td></td>
<td>11.35</td>
<td>37.2</td>
</tr>
<tr>
<td>Shear modulus</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>G12</td>
<td>GPa</td>
<td>29.27</td>
<td>2.4</td>
</tr>
<tr>
<td>G23</td>
<td></td>
<td>29.27</td>
<td>2.4</td>
</tr>
<tr>
<td>Poisson’s ration</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>( \nu_{c12} )</td>
<td></td>
<td>0.29</td>
<td>0.29</td>
</tr>
<tr>
<td>( \nu_{c23} )</td>
<td></td>
<td>0.035</td>
<td>0.29</td>
</tr>
<tr>
<td>Density</td>
<td>kg·m(^{-3})</td>
<td>1478</td>
<td>1786</td>
</tr>
</tbody>
</table>

### Results and discussion

In this test 6 variants of HE100B and its composite analogues were tested. The I-beam was simulated using a curvature-based mesh with 10.67 mm largest element size and 3.55 mm smallest element size configuration. Maximal values of the normal stress at the stress force applying points are neglected and filtered from the results by Saint-Venant’s principle [25]. The deflection along Y axis is denoted as \( u_y \) and the normal stress along Z axis is denoted as \( \sigma_z \). The main interest is focused on the deflection rate \( u_y \).

### Table 2

<table>
<thead>
<tr>
<th>Beam number</th>
<th>I-beam</th>
<th>( u_{y\text{max}}, \text{mm} )</th>
<th>( \sigma_{z\text{max}}, \text{N}\cdot\text{m}^{-2} )</th>
</tr>
</thead>
<tbody>
<tr>
<td>1</td>
<td>S235JR</td>
<td>-1.47E + 00</td>
<td>8.91E + 07</td>
</tr>
<tr>
<td>2</td>
<td>GF40%</td>
<td>-8.36E + 00</td>
<td>8.91E + 07</td>
</tr>
<tr>
<td>3</td>
<td>CF40%</td>
<td>-3.46E + 00</td>
<td>8.98E + 07</td>
</tr>
<tr>
<td>4</td>
<td>CFLW40% Unidirectional laminas</td>
<td>-5.29E + 00</td>
<td>1.17E + 08</td>
</tr>
<tr>
<td>5</td>
<td>CFLW40% Bidirectional laminas</td>
<td>-5.94E + 00</td>
<td>1.36E + 08</td>
</tr>
<tr>
<td>6</td>
<td>CF40% + CFLW Bidirectional laminas</td>
<td>-4.10E + 00</td>
<td>1.00E + 08</td>
</tr>
</tbody>
</table>

The beam number 1 is the etalon beam. Deflection rate of the beam which is closer to the deflection rate of the beam number 1 is counted as the optimal option. All of the FRP beams are lighter than the steel beam. According to the results obtained by the finite element method the closest to the I-beam number 1 in terms of the deflection is the I-beam number 3. But still there is a large difference comparing to the steel I-beam (number 1), 1.47 mm for the beam number 1 and 3.46 mm for the beam number 3. Perhaps, the deflection rate could be decreased by increasing the reinforcement ratio from 40% to 60% and more. Glass fibre I-beam (number 2) has the largest deflection rate. The normal stresses shown are the normal stresses at the top flange of the I-beam. According to the materials properties [14-19] the normal stresses are acceptable. It opens new opportunities for future tests and experiments of new configurations of the reinforcement materials especially in the I-beam cross-section shape. So,
considering the mass difference, the obtained results could be used for finding optimal solutions depending on the purpose of the I-beam.

![Normal stress in Z axis](image1)

![Deflection in Y axis](image2)

**Fig. 3. Normal stress in Z axis**

**Fig. 4. Deflection in Y axis**

**Conclusions**

According to the results obtained by the finite element method the analysis showed results in favour of the unidirectional reinforcement rather than layered reinforcement. Carbon fibre I-beams are closer to steel by the elastic properties but much lighter. One cubic meter of 40% carbon fibre reinforced polymer weights 1478 kg (found using the rule of mixture) but 1 cubic meter of steel weights 7800 kg. Almost 5.28 times 40% reinforced carbon fibre is lighter than steel. This opens new opportunities for I-beams to be used for different architectural application due to the weight. Perhaps, carbon fibre I-beams with larger reinforcement ratio could show the desired mechanical behaviour with less deflection. For further and more detailed analysis experiments should be conducted. Glass fibre I-beams could be used for less loaded application rather than carbon fibre I-beams. Carbon fibre I-beams can be an alternative for steel I-beams.

**Author contributions**

Writing, editing, simulation and conceptualization I. Jafarli; CAD modelling and software consulting U.H. Vavaliya. All authors have read and agreed to the published version of the manuscript.

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